

Matteo Riboli
R&D Engineer
mriboli@flowserve.com
+3905235767213

Limitorque Fluid Power System





Fatigue assessment of scotch yoke for high cycles applications in heavy-duty valves actuators



Preliminar consideration

Actuator performance in high cycles applications: trust or verify?

 Actuator life-span is typically measured in cycles of operations – cycle life.

 Many flow-control processes require on-off valves and actuators to cycle with great frequency, meaning a reduced temporal lifespan and increased maintenance intervals.

 Very often valves and actuators cycle life is based on estimation or field history and is not validated rigorous experimental evidence.



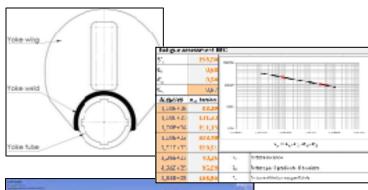


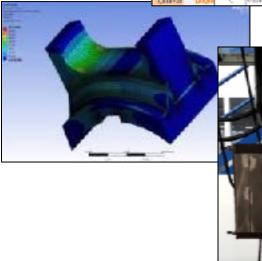
Challenge and applied Methodologies

Analytical analysis

Numerical analysis

Experimental test





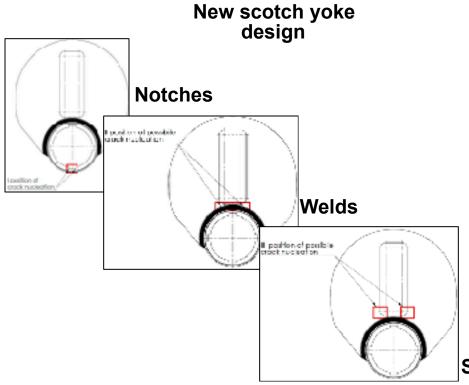




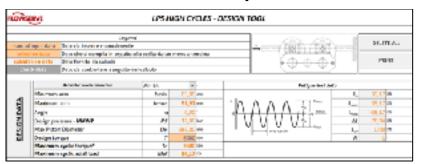
Analytical analysis



Automatic spreadsheet



Link between spreadsheet and 3D template



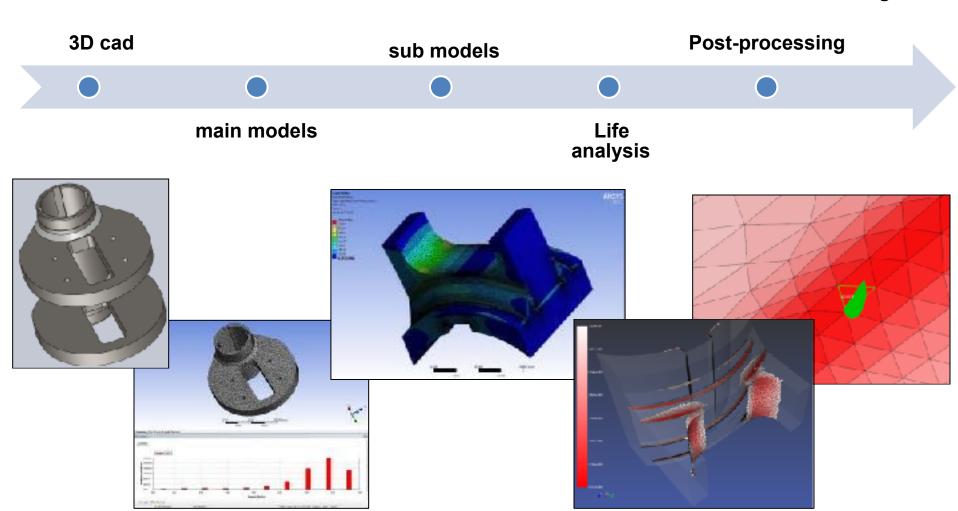
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Slider Surface





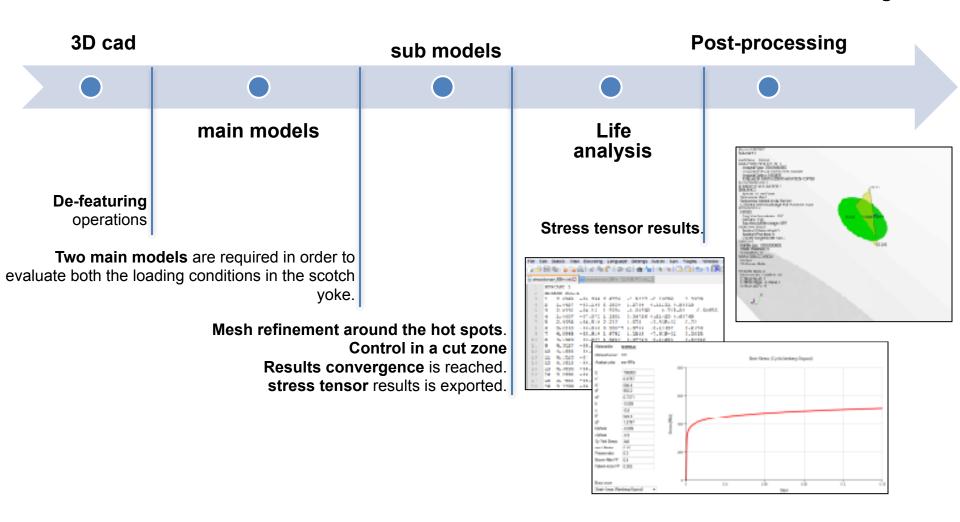
Numerical analysis







Numerical analysis







Numerical analysis





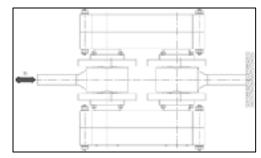


Experimental test

Set-up definition

Tool design





• Fatigue test

IV

Output data analysis

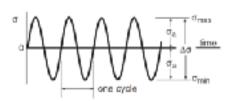






Experimental test

Set-up definition

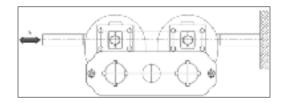


$$R = \frac{\sigma_{min}}{\sigma_{max}} = -1$$

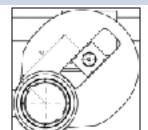
Frequencies to be define base on the noise in the laboratory.

Tool design

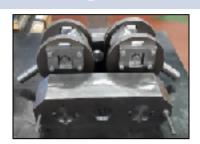
Defining of b, position of the slider blocks



Tool realization, and premounting



Tool design maintaining the same b



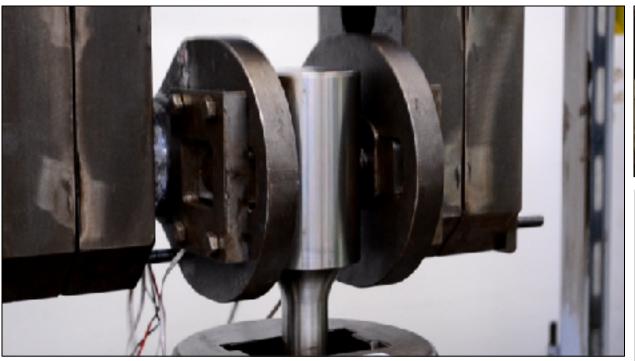




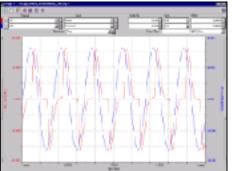
Experimental test

III

Fatigue test







Finished the tool and samples assembly on the MTS, the fatigue test on two yokes is started.





Experimental test



Fatigue test



Position I 8,59 · 10⁵ cycles MTT = 6000Nm



Position II $8,06 \cdot 10^{6}$ cycles MTT = 6000Nm



Position III 6,03 · 10⁵ cycles NTT = 5900Nn

ID				cycles	Crack positions	Note
1	130	10000	6000	53000	I	DISCARDED
2	130	10000	6000	859000	I	1
3	90	7000	4200	Run-out	1	1
4	90	7000	4200	Run-out	1	1
3	200	15400	9100	85000	II – III	1
4	200	15400	9100	85000	II – III	1
5	130	10000	6000	806000	III	1
6	150	11500	6900	287000	I — II — III	1
7	150	11500	6900	603000	II – III	1
8	150	11500	6900	316000	III	1
9	180	13800	8300	120000	I – II – III	1
10	180	13800	8300	120000	I — II — III	1
11	130	10000	6000	661000	II – III	1
12 MCT	130	10000	6000	661000	II — III	1

MST – Maximum structural torque;

MTT – Maximum test torque*.





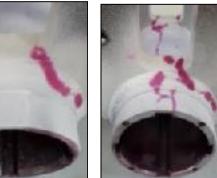
Experimental test

IV

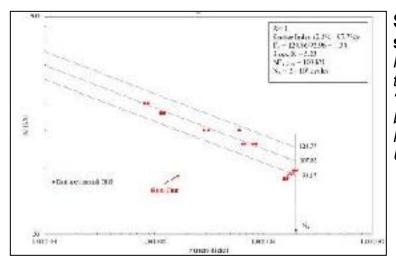
Output data and results





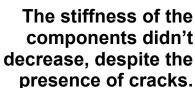


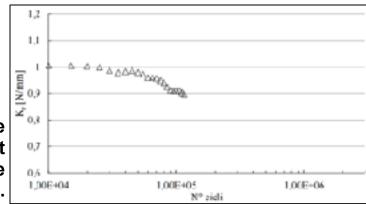
Crack position highlighted by PT test.



S/N curve of the scotch yoke. Picture obtained thanks the software

thanks the software "Diciotto", developed by Prof.Eng. Giovanni Meneghetti of the University of Padova.









Results comparison







Conclusions



 From the experimental test conducted in the University of Padova the most critical point of the scotch yoke seems the position 3, the final part of the sliding surface.



 Despite the presence of some cracks in the sample yoke, the component's stiffness not decreases. The yoke continues to work without problems despite the cracks.



Future Developments

Ĭ

 Obtain a numerical feedback that the same loading conditions which we can find in a real application are recreate in the laboratory test conducted in this step.

II

 Verify if the surface treatment applied at the scotch yoke influence or not the fatigue behavior highlighted in this first project step.



 Some other tests on e new design of scotch yokes are foresee in order to optimize the design with some other adjustments.





Acknowledgments

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- **Eng. Matteo Bergonzoni** Scholarship holder University of Padova.

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Not last...

thanks to all for your attention!!!







